

# Frequency based design methods



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# Bode's graph based design

- By shaping frequency response of open loop (OL) – “easy” – indirectly shaping the closed loop (CL) – the aim
- Typically we tune values like  $\omega_C$ , 0dB crossing,  $PM$ ,  $BW$ ,  $GM$

$$L(j\omega) = G(j\omega)D_C(j\omega) \quad \rightarrow \quad \begin{aligned} |L(j\omega)|_{\text{dB}} &= |G(j\omega)|_{\text{dB}} + |D_C(j\omega)|_{\text{dB}} \\ \arg(L(j\omega)) &= \arg(G(j\omega)) + \arg(D_C(j\omega)) \end{aligned}$$

- Disadvantages: practically we are not able to detect instability, it must be guaranteed from “outside” – these methods can't be used for stabilization of unstable systems – therefore these methods are used mainly/exclusively for stable (usually minimum phase) systems => also CL is stable (according to Nyquist's criterion)
- Not show stopper, representation of Bode's graph for unstable system is questionable (It's not possible to measure, formally it is possible to draw it,...)
- Frequency based method are suited for measured frequency response: We can do the graphical design without mathematical model
- The time delay is not issue for such a design



# Gain for required study state behavior

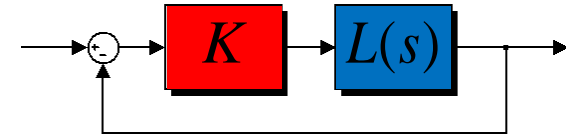
**Aim:** Tune control law gain in such a way, that study state error (for step command) is lower than preselected value. Usually expressed by position constant  $K_p$

1. Measure/read “initial” study gain
2. The different between required and measured gain is control law gain.

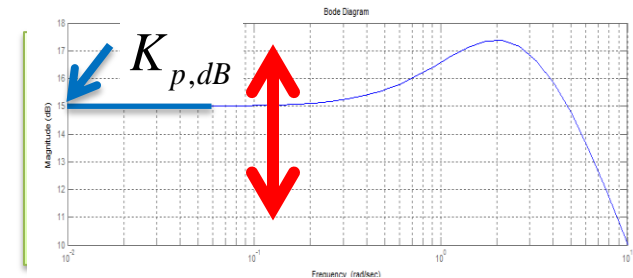
**Aim:** Tune control law gain in such a way, that study state error (for ramp command) is lower than preselected value. Usually expressed by velocity constant  $K_v$

1. Compute required crossing point of low frequency asymptote with „0dB level“
2. Tune the gain to shift the frequency respons towards the crossing point

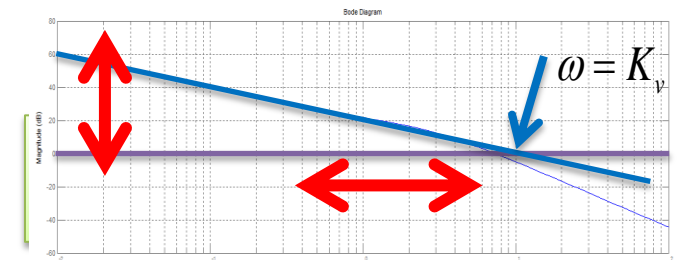
**Check CL Stability!**



$$e_{\text{step,ss}} = 1/(1 + K_p)$$



$$e_{\text{ramp,ss}} = 1/K_v$$



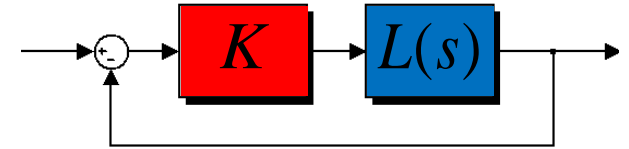


# Control law gain tuning for required $PM$

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Aim:

Tune control law in such a way to ensure CL  $Phase\ Margin$  (thereby overshoot, damping)

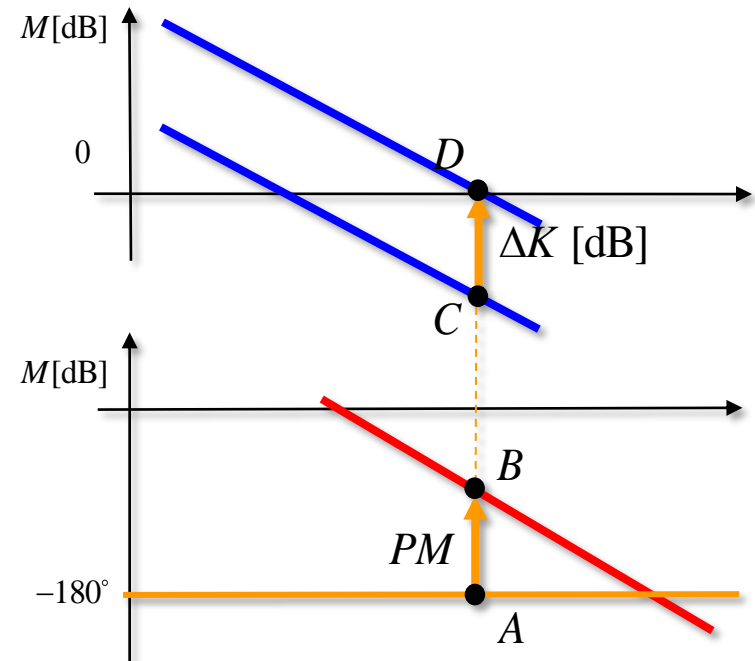


$$PM = \arctan \frac{2\zeta}{\sqrt{-2\zeta^2 + \sqrt{1+4\zeta^4}}}, \zeta = \frac{-\ln(\%OS/100)}{\sqrt{\pi^2 + \ln^2(\%OS/100)}}$$

Algorithm (for sable L!)

1. Define  $PM$ , or compute it form  $\%OS$  or  $\zeta$
2. Draw Bode's graph for arbitrary selected  $K$
3. Find  $\omega_{PM}$  frequency on phase graph, where phase is equal to  $-180^\circ + \text{required } PM$
4. Change gain in such a way that amplitude for particular frequency is equal to 1 (0 dB), so the

$$\omega_c = \omega_{PM}$$





# PI control law tuning

## PI – Proportional – Integral controller

$$D_C(s) = K_P + \frac{K_I}{s} = K_P \left( 1 + \frac{K_I}{sK_P} \right)$$

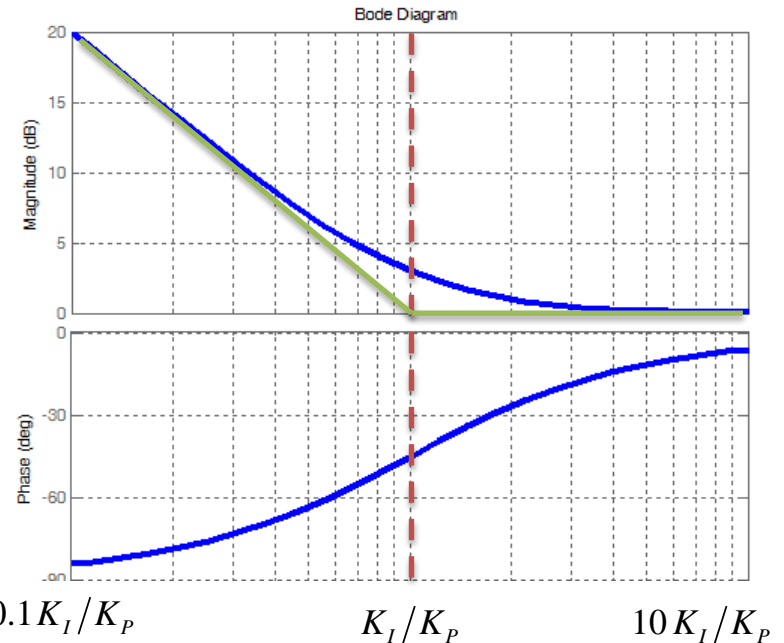
- Low pass filter:

$$|D_C(j\infty)|_{\text{dB}} = 20 \log K_P < 0 \quad \text{pro } K_P < 1$$

- Design is done by tuning  $K_P$  in such a way that new  $\omega_c$  ensure better  $PM$
- The cut-off frequency is then determined by gain  $K_I$ . Proper choice is at the “left”, to avoid phase lag around  $\omega_c$ .

$$K_I / K_P = \omega_{c,\text{new}} / 10$$

- PI increase  $T_r$ ,  $GM$ ,  $PM$  and  $M_r$  and decrease  $\omega_{BW}$ , and  $OS$
- Advantage is high frequency noise attenuation.





## PD – Proportional – derivative controller

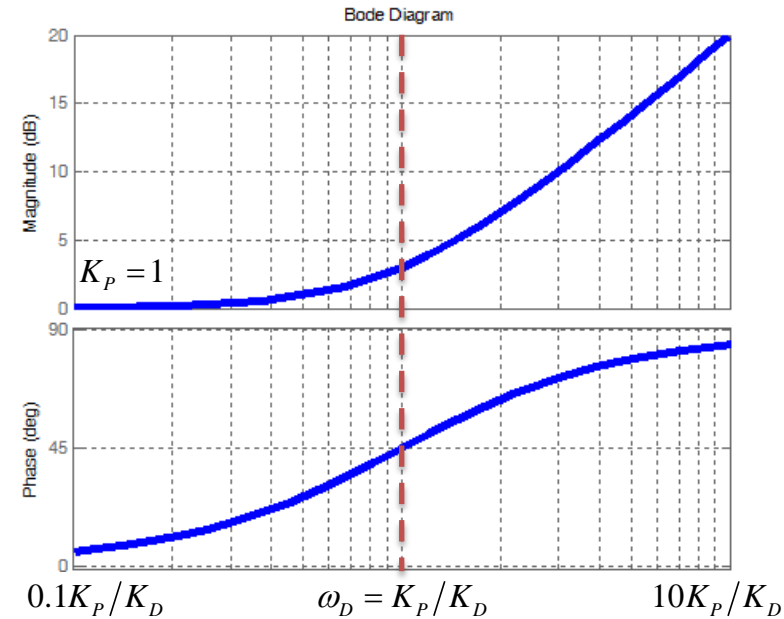
$$D_C(s) = K_P + K_D s = K_P \left( 1 + \frac{K_D}{K_P} s \right)$$

- High pass filter
- Increase phase on higher frequencies (phase lead), is used for increase of  $PM$ ,
- also increase gain on higher frequencies, thereby shifting  $\omega_c$
- Cut-off frequency is selected by gains:

$$\omega_D = K_P / K_D$$

in such a way that  $PM$  is increased.

- PD controller increase  $\omega_{BW}$ ,  $GM$  and  $M_r$  and decrease  $T_r$ , and  $OS$ .
- Disadvantage is high frequency noise magnification
- Issue: practical realization is slightly different (realization of derivative).



$$\varphi(\omega_D) = 45^\circ$$

$$|D_C(j\omega_D)| = K_D \omega_D \sqrt{2}$$



PID – Proportional – integral – derivative controller

$$D_C(s) = K_P + \frac{K_I}{s} + K_D s = (1 + K_{D1}s) \left( K_{P2} + \frac{K_{I2}}{s} \right)$$

$$K_P = K_{P2} + K_{D1}K_{I2}$$

$$K_D = K_{D1}K_{P2}$$

$$K = K_{I2}$$

Design

- First design PD part then PI part
- Or vice versa



# Lag compensation for required $PM$ and $K_v$

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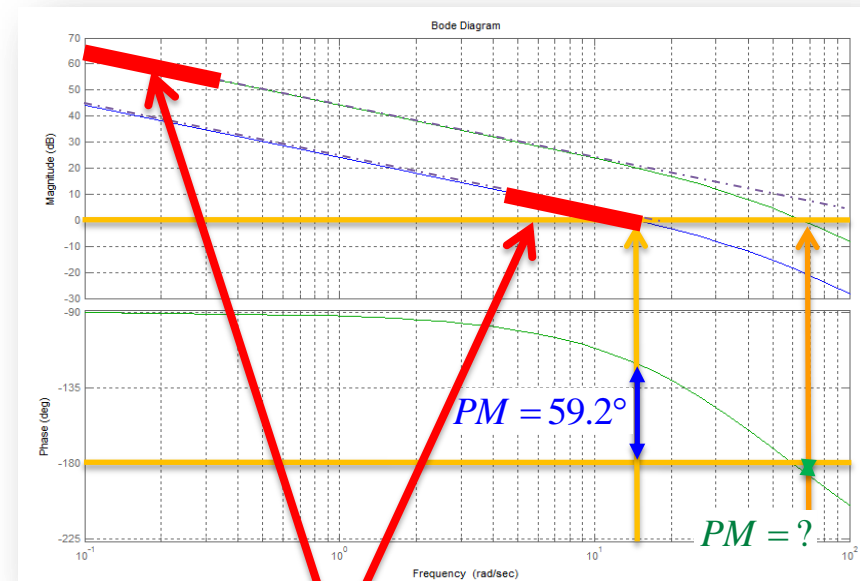
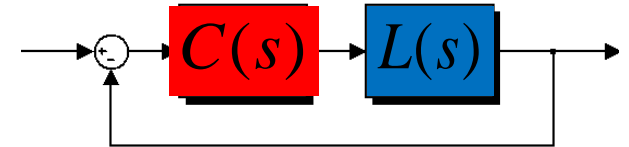
**Aim:** Ensure required  $PM$  and  $K_v$

**Discussion:**

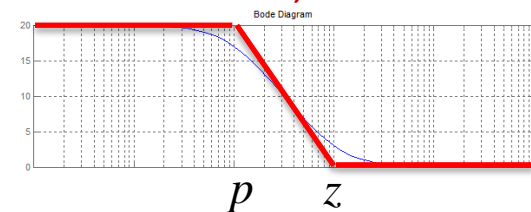
- Required  $K_v$  ensure gain according to green graph, but it decrease  $PM$
- On the other hand by decreasing gain (modrý graph), we increase  $PM$ , but decrease  $K_v$  and thereby we increase ramp tracking error

**Conclusion:**

- It is not possible to ensure both just by gain
- „lag“ type controller is required



Perfect solution, how to ensure it ?







# Lag controller

## Lag controller (with phase lag)

- Transfer function  $D_C(s)$ , or alternatively  $C(s)$

$$D_C(s) = K \frac{s+z}{s+p}, p < z$$

$$C(s) = \alpha \frac{Ts+1}{\alpha Ts+1} = \frac{s+\frac{1}{T}}{s+\frac{1}{\alpha T}}, \alpha > 1$$

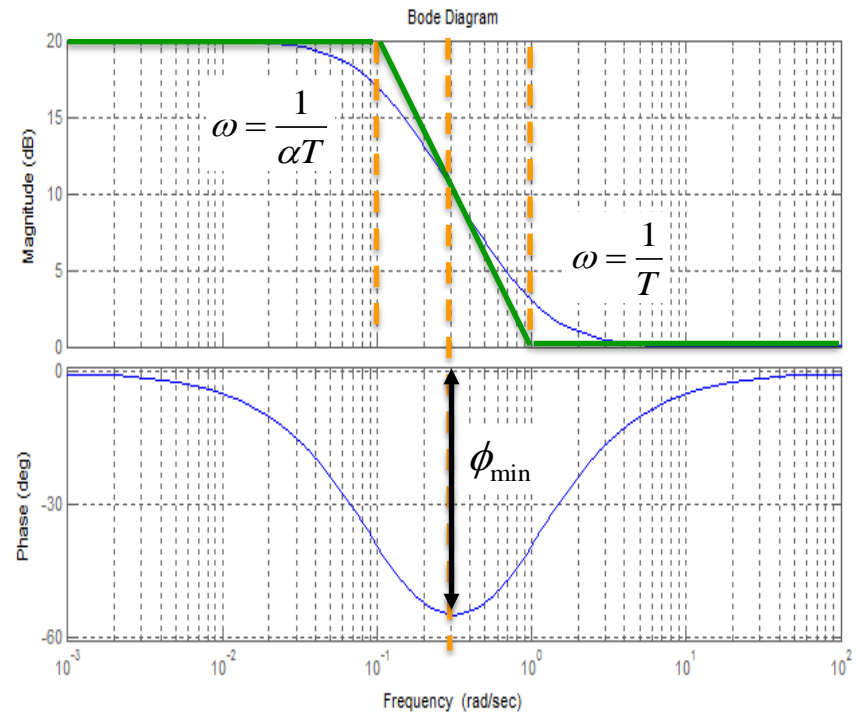
where

$$D_C(s) = KC(s), z = 1/T, p = 1/\alpha T$$

- Frequency response

Can be used for:

- Improving steady state error by increasing of gain just on low frequencies



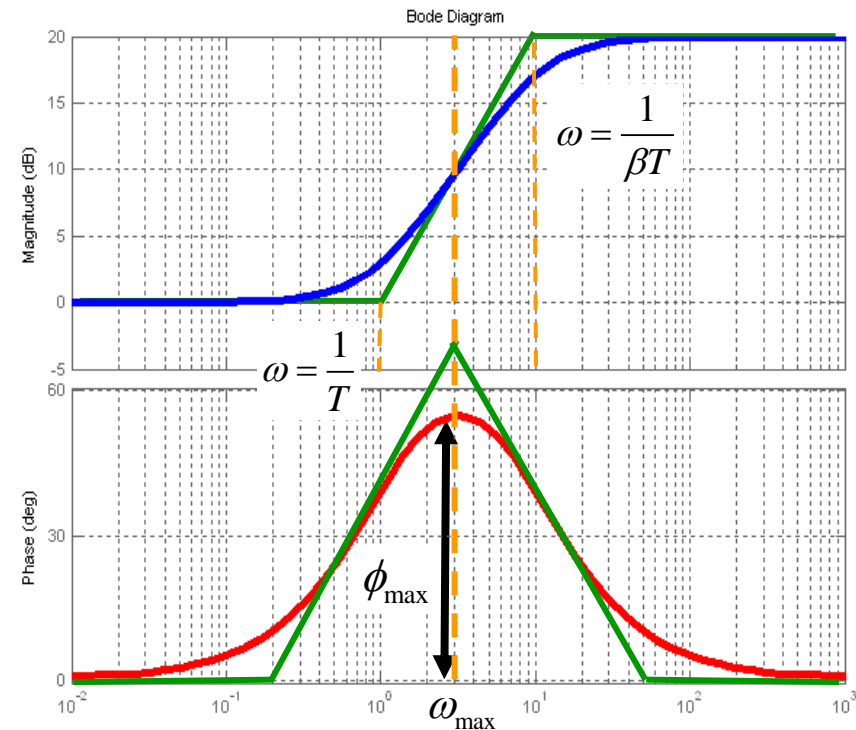


# Lead compensator

## Lead compensator

$$D(s) = \frac{Ts + 1}{\beta Ts + 1} = \frac{1}{\beta} \frac{s + \frac{1}{T}}{s + \frac{1}{\beta T}}, \quad \beta < 1$$

- in this case the DC gain is =1, to preserve already designed study state behavior constant.
- Comparing with PD it has finite gain on high frequencies





- Let derive equations needed for design

- The phase of controller is

$$\phi = \arctan \omega T - \arctan \omega \beta T$$

- Let derive by  $\omega$

$$\frac{d\phi}{d\omega} = \frac{T}{1+(\omega T)^2} - \frac{\beta T}{1+(\omega \beta T)^2} \stackrel{!}{=} 0$$

- Phase is maximal at frequency  $\omega_{\max} = \frac{1}{T\sqrt{\beta}}$  for which  $D(j\omega_{\max}) = \frac{j\frac{1}{\sqrt{\beta}} + 1}{j\sqrt{\beta} + 1}$

- The relationship between phase and  $\beta$  is

$$\phi_{\max} = \arctan \frac{1-\beta}{2\sqrt{\beta}} = \arcsin \frac{1-\beta}{1+\beta} \quad \text{ans vice versa} \quad \beta = \frac{1-\sin \phi_{\max}}{1+\sin \phi_{\max}}$$

$$|D(\omega_{\max})| = 1/\sqrt{\beta}$$

